

TASK: 1

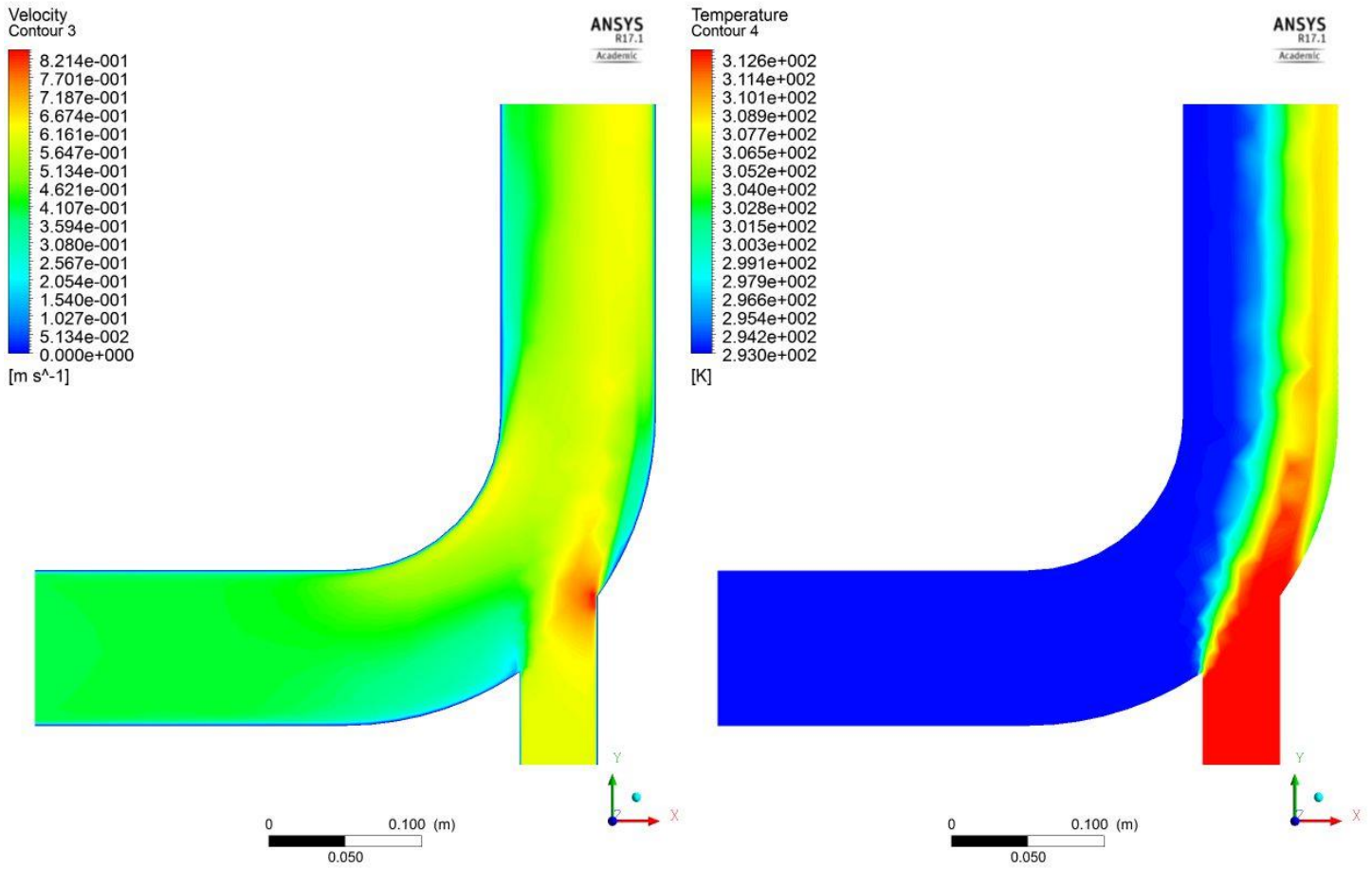


Figure 1: Counterplot of velocity and temperature contour plots along the plane of symmetry

The above contour plots have been obtained after increasing the radius of small inlet of the pipe from 0.75 in in tutorial to 1 inch and velocity of small pipe inlet has been reduced from 1.2m/sec to 0.6 m/sec. The simulation was run for 500 iterations. Second order upwind in the solution method has been used to calculate the velocity and temperature at the plane of symmetry.

Temperature
Contour 3

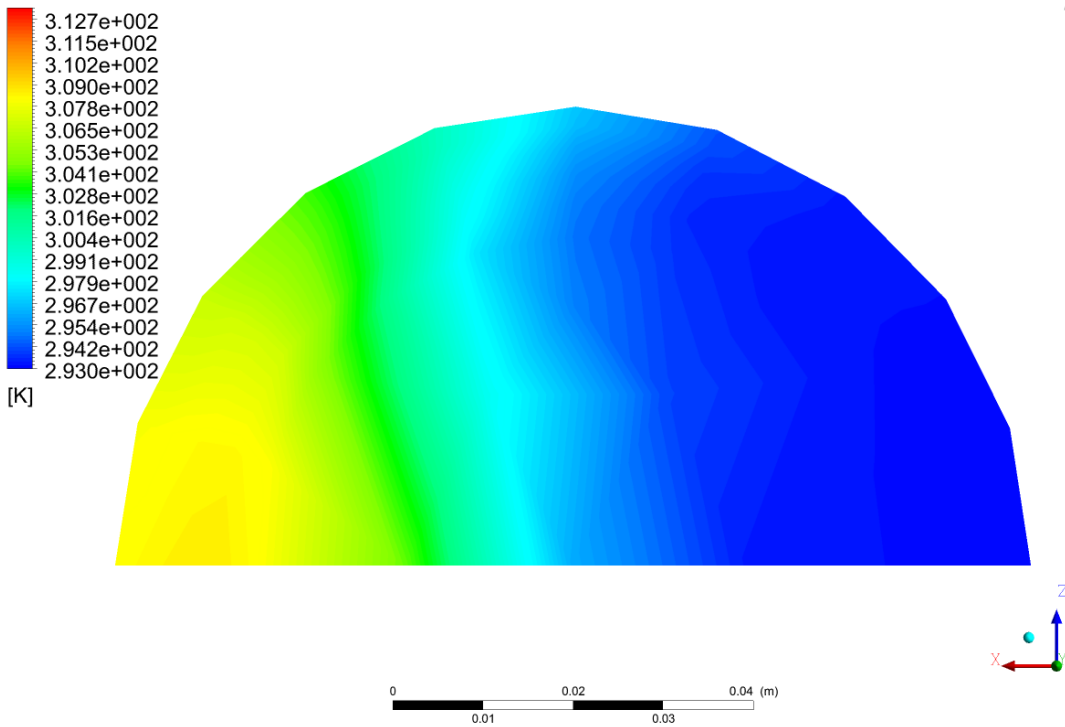


Figure 2: Temperature contour plot for large pipe outlet

The above plot has been obtained by selecting the pressure outlet surface of the pipe and the temperature has been calculated for 500 iterations. Second order upwind solution method has been used to calculate the temperature at the outlet.

TASK: 2

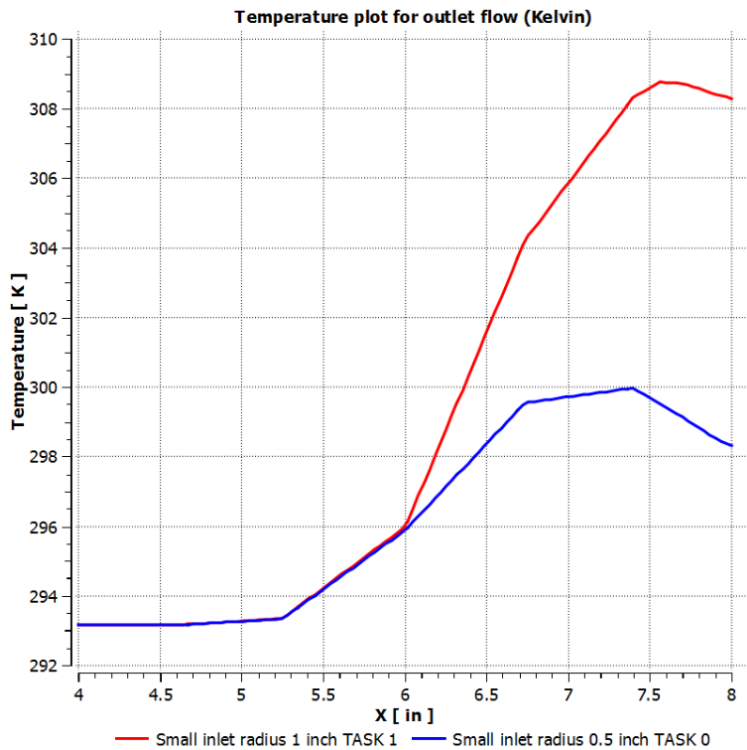


Figure 3: Temperature line plot normal to outlet along AB

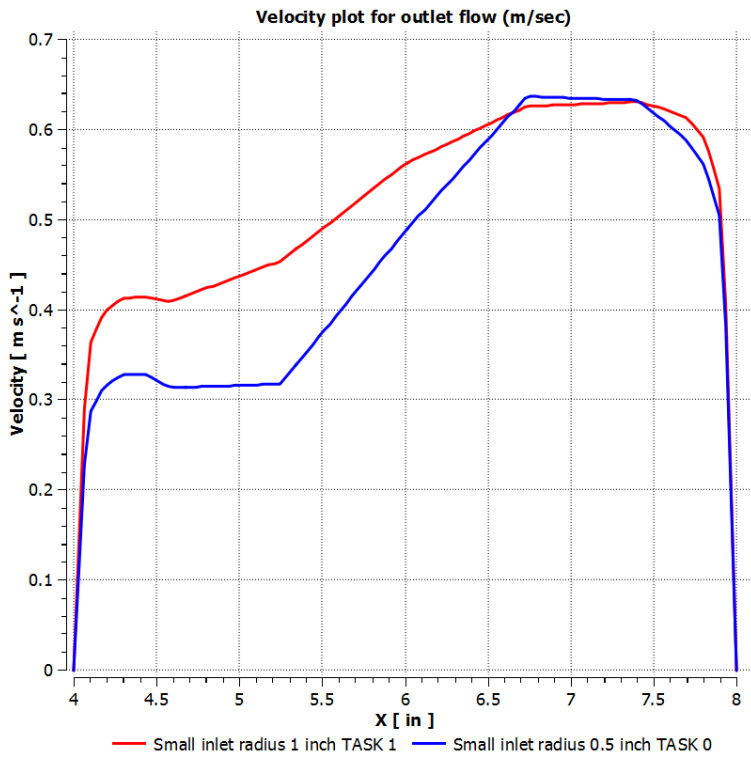


Figure 4: Velocity line plot normal to outlet along line AB

TASK: 3

"Surface Integral Report"

Integral	
custom-function-1	

pressure-outlet	2756760.5
Integral	
custom-function-1	

velocity-inlet-large	1971182.2
velocity-inlet-small	789624.82

Net	2760807

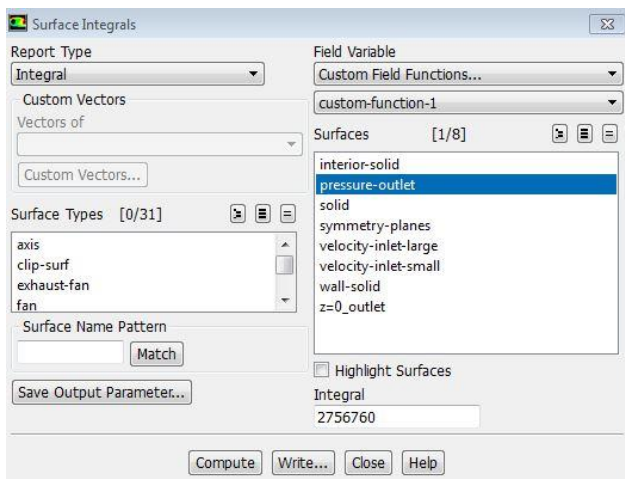
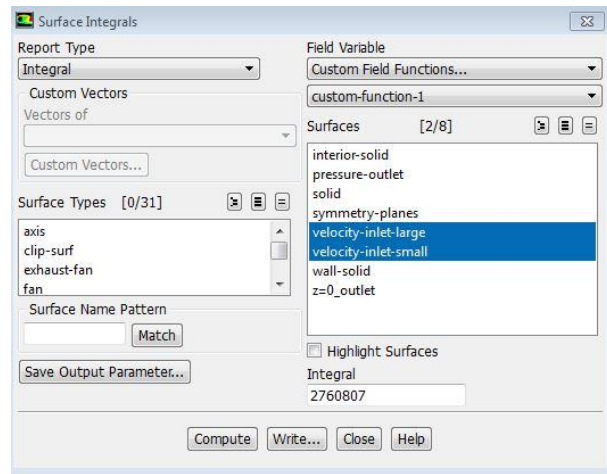


Figure 5: Flow of heat comparison over inlet and outlet using fluent custom field function

"Surface Integral Report"	
Integral mass	
pressure-outlet	2.1898614
Integral mass	
velocity-inlet-large	1.5949103
velocity-inlet-small	0.59809168
Net	2.193002

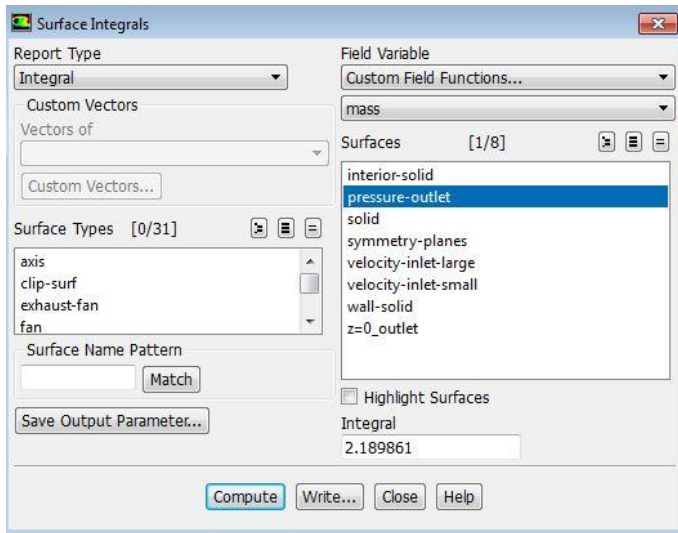
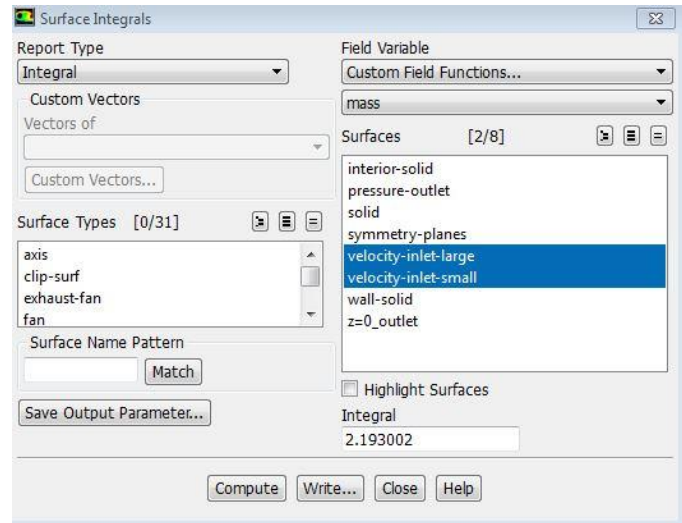


Figure 6: Mass flow comparison over inlet and outlet using fluent custom field

Flow rate of heat through a given surface A is given by:

$$H = \iint Vn \times \rho \times Cp \times T \times dA \quad (1)$$

When we substitute for H in the above equation we get

$$H = 0.6 \times 1000 \times 4216 \times 313 \times 0.0254 \times 0.0254 \times 3.14$$

H=802386.2917 J/sec at the small pipe inlet. The value calculated is very close to the value calculated by fluent which 789624.82 J/sec.